

BELOW-THE-HOOK LIFTING DEVICE
Engineering Note Cover Page

Lifting Device Numbers: 173A
FNAL Site No/ _____ Div. Specific No. MD-196A Asset No. _____
If applicable If applicable If applicable

- ASME B30.20 Group: (check one)
- Group I Structural and Mechanical Lifting Devices
 - Group II Vacuum Lifting Devices
 - Group III Magnets, Close Proximity Operated
 - Group IV Magnets, Remote Operated

Device Name or Description GAS CYLINDER LIFTING FIXTURE

- Device was (check all applicable)
- Purchased from a Commercial Lifting Device Manufacturer. Mfg Name _____
 - Designed and Built at Fermilab
 - Designed by Fermilab and Built by a Vendor. Assy drawing number _____
 - Provided by a User or other Laboratory
 - Other: Describe _____

Engineering Note Prepared by Mike Zuckaast Date 28 July 2009
Engineering Note Reviewed by ANG LEE Date 4 Aug 2009

Lifting Device Data:

Capacity 1200 #
Fixture Weight 100 #
Service: normal heavy severe (refer to B30.20 for definitions)

Duty Cycle _____ 8, 16 or 24 hour rating (applicable to groups III, and IV)

Inspections Frequency EACH USE
Rated Load Test by FNAL (if applicable) Date 24 FEB 2010 Load 2000 #

Check if Load Test was by Vendor and attach the certificate

Satisfactory Load Test Witnessed by: TIM GRIFFIN
Signature (of Load Test Witness) [Signature] 5108

Notes or Special Information:

BELOW-THE-HOOK LIFTING DEVICE
Engineering Note Cover Page

Lifting Device Numbers: 173B

FNAL Site No/ _____ Div. Specific No. MD 196B Asset No. _____
 If applicable If applicable If applicable

ASME B30.20 Group: Group I Structural and Mechanical Lifting Devices
 (check one) Group II Vacuum Lifting Devices
 Group III Magnets, Close Proximity Operated
 Group IV Magnets, Remote Operated

Device Name or Description GAS CYLINDER LIFTING FIXTURE

Device was Purchased from a Commercial Lifting Device Manufacturer. Mfg Name _____
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Notes or Special Information:



Fermilab

Particle Physics Division

Mechanical Department Engineering Note

Number: MD-ENG-196

Date: 28 July 2009

Project Internal Reference:

Project:

Title: Gas Cylinder Lifting Fixture

Author(s): Mike Zuckerbrot

Reviewer(s): *Anghel Aug 4, 2009*

Key Words: Lifting fixture, gas cylinder

Applicable Codes:

Abstract Summary:

The following note analyzes the capacity of a gas cylinder lifting fixture; considering loading with either two or four cylinders present.



Picture: Actual gas cylinder lifting fixture for analysis

Discussion

- $\sigma_y = 30 \text{ ksi}$
- Will consider loading with 2 and 4 tanks
- Assume standard type K cylinder size filled with water

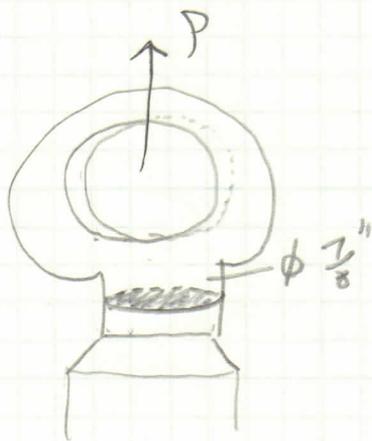
→ 9.25×60 " ; weights 135 lbs
 (d) (h)
 Water capacity = 110 lbs

→ Assume each tank = 300 lbs

| Tanks | P_{Total} (#) |
|-------|------------------------|
| 2 | 600 |
| 4 | 1200 |

Eye

- Tensile stress



$\sigma_{\text{max}} = \frac{1}{3} \sigma_y = 10 \text{ ksi}$

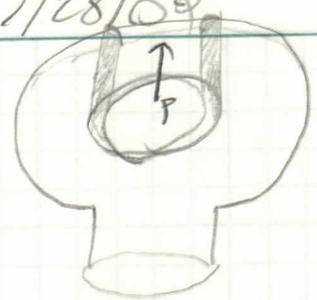
$\sigma = \frac{P}{A}$; $A = \frac{1}{4} \pi \left(\frac{7}{8}\right)^2 = .601 \text{ in}^2$

| Tanks | σ_{Total} (psi) | |
|-------|-------------------------------|------------------------------|
| 2 | 998 | $< 10 \text{ ksi}$ <u>ok</u> |
| 4 | 1997 | |

shear stress

$$\sigma_{max} = \frac{1}{3} F_y = 10 \text{ ksi}; \quad \sigma = \frac{P}{A}$$

$$A = 2 \left[\frac{1}{4} \pi \left(\frac{7}{8} \right)^2 \right] = 1.203 \text{ in}^2$$

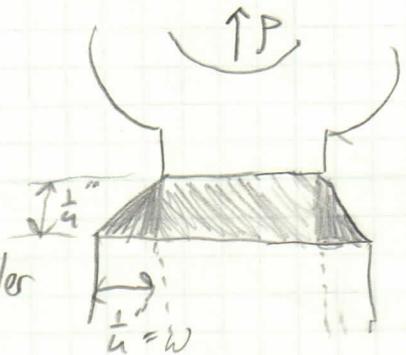


| Tanks | σ_{Total} (psi) |
|-------|------------------------|
| 2 | 499 |
| 4 | 998 |

< 10 ksi ok

Eye weld stress

$$\begin{aligned} \sigma_{max} &= \sigma_y (0.3) \rightarrow \text{assume E60xx filler} \\ &= (60 \text{ ksi}) (0.3) = 18 \text{ ksi} \end{aligned}$$



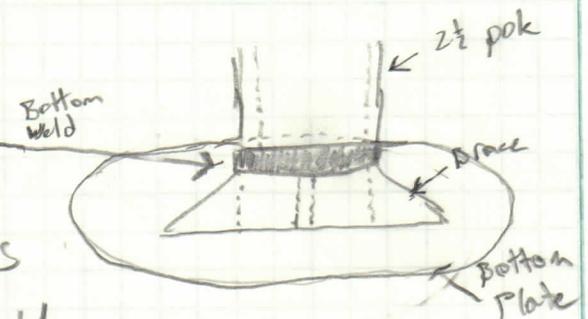
$$\text{Throat Area} = (0.707) \left(\frac{1}{4} \right) \times \pi \left(\frac{7}{8} \right) = 0.486 \text{ in}^2$$

| Tanks | σ_{Total} (psi) |
|-------|------------------------|
| 2 | 1235 |
| 4 | 2469 |

< 18 ksi ok

Bottom weld stress

→ The bottom weld has greater area than the eye weld, therefore



$$\sigma_{Bottom \text{ weld}} < \sigma_{Eye \text{ weld}} < 18 \text{ ksi} \quad \underline{ok}$$

Bottom Plate

→ Weld stress

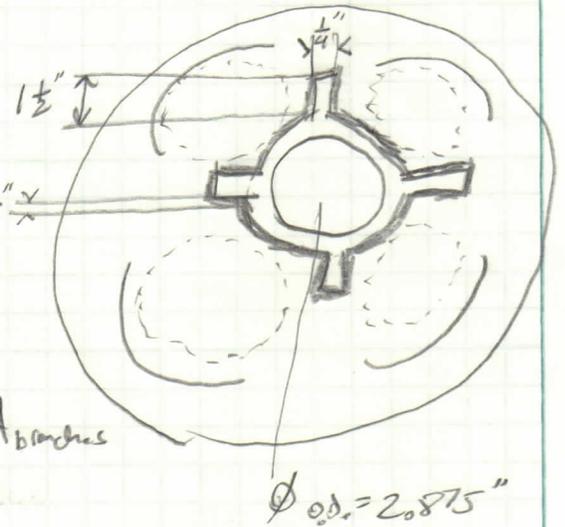
$$\sigma_{max} = (0.3) F_y = 18 \text{ ksi}$$

$$\sigma = \frac{P}{A} ; A_{Total} = A_{pipe} + A_{branches}$$

$$A_{pipe} = 0.707 \pi d w = 0.707 \pi (2.875) \frac{1}{4} = 1.6 \text{ in}^2$$

$$A_{branches} = 0.707 \left(\frac{1}{4}\right) \left(1\frac{1}{2}\right) \times (8 \text{ sides}) = 2.121$$

$$A_{Total} = 1.6 + 2.121 = 3.721 \text{ in}^2$$



| Tanks | σ_{Total} (psi) |
|-------|------------------------|
| 2 | 161 |
| 4 | 322 |

< 18 ksi ok

— Bending stresses on the Bottom Plate are considered in the FEA seen on the next pages

FEA

| Tanks | max δ (in) | Max σ (localized at intersection of brace corners and plate) (psi) |
|-------|-------------------|---|
| 2 | 0.0102 | 7,150 |
| 4 | 0.0108 | 11,100 |

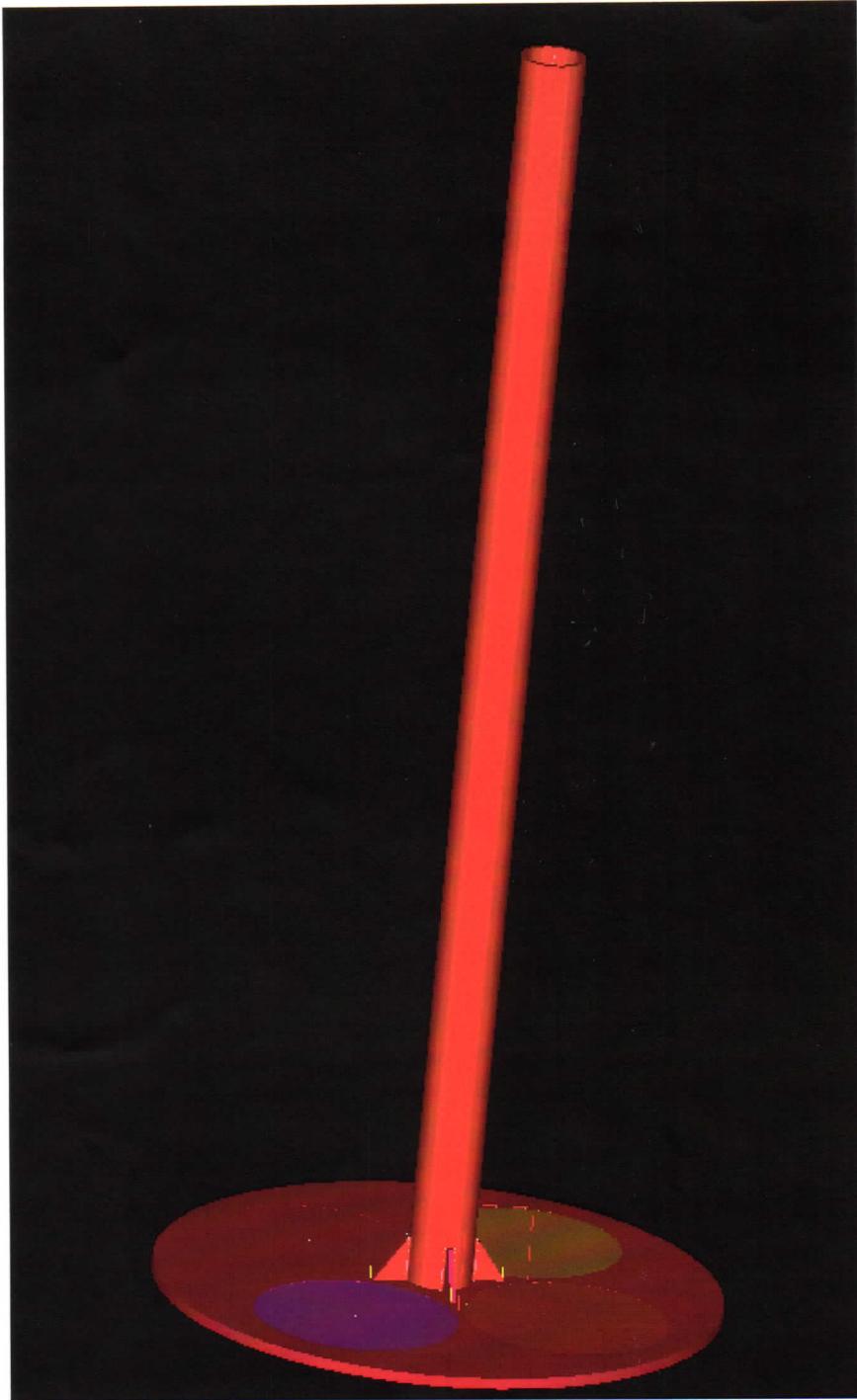


Figure: Equivalent fixture modeled in I-DEAS

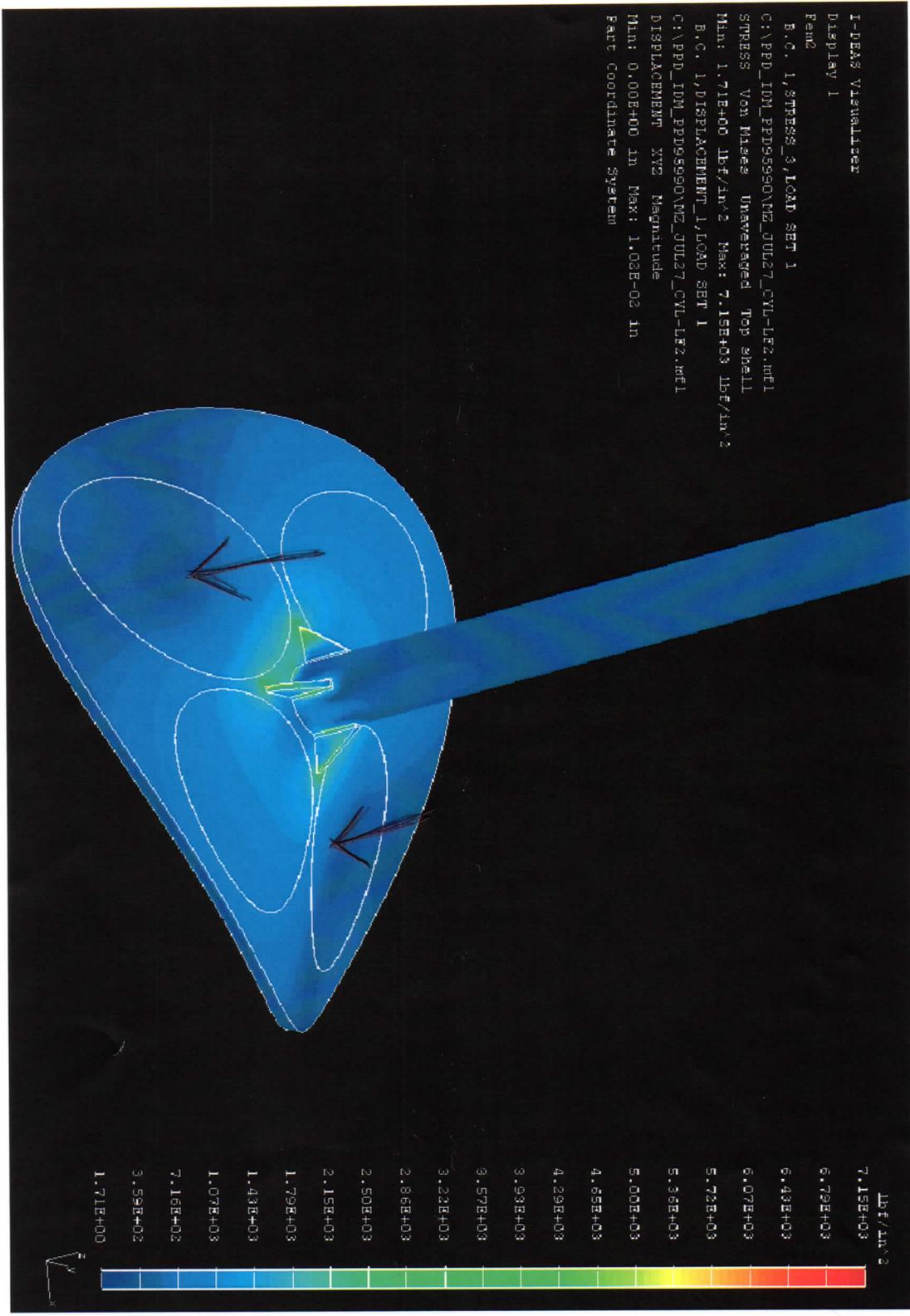


Figure: FEA with 2-300lb gas cylinders present; max stress is 7150 psi at very localized points where the corners of the brace meet the bottom plate (welds, Fmax = 18ksi).

* Tanks always opposite each other when only Z present

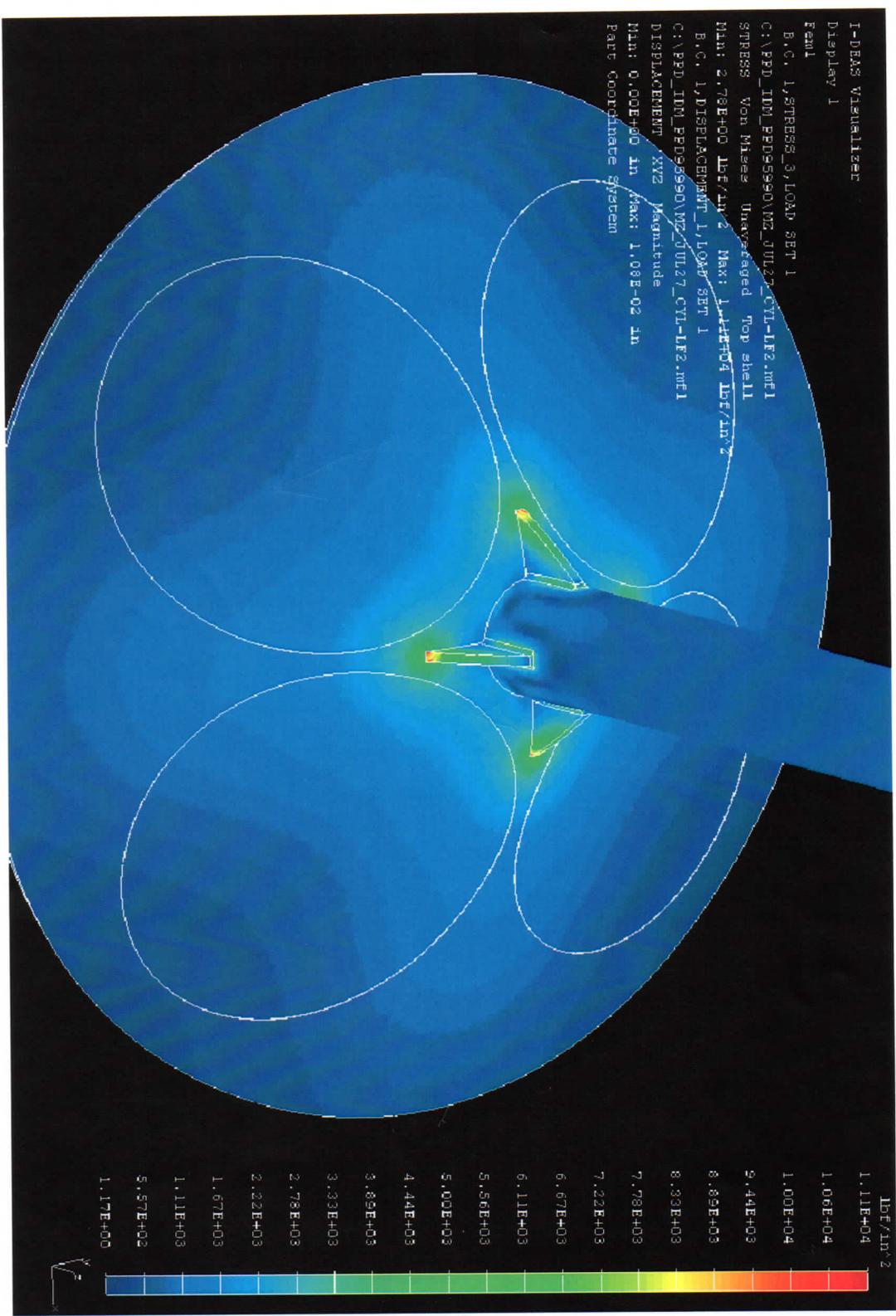


Figure: FEA with 4-300 lb cylinders presents; max stress is 11100 psi at very localized points where the corners of the brace meet the bottom plate (welds, Fmax = 18ksi).